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OPTIMIZED INNER BOUNDARY SHAPES IN CIRCULAR RINGS UNDER DIAMETR--ETC(U)

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OPTIMIZED INNER BOUNDARY SHAPES IN CIRCULAR RINGS
UNDER DIAMETRAL COMPRESSION

BY

A. J. DURELLI AND K. RAJATAH

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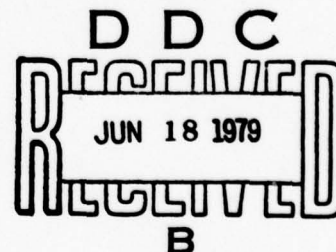
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Previous Technical Reports to the Office of Naval Research

1. A. J. Durelli, "Development of Experimental Stress Analysis Methods to Determine Stresses and Strains in Solid Propellant Grains"--June 1962. Developments in the manufacturing of grain-propellant models are reported. Two methods are given: a) cementing routed layers and b) casting.
2. A. J. Durelli and V. J. Parks, "New Method to Determine Restrained Shrinkage Stresses in Propellant Grain Models"--October 1962. The birefringence exhibited in the curing process of a partially restrained polyurethane rubber is used to determine the stress associated with restrained shrinkage in models of solid propellant grains partially bonded to the case.
3. A. J. Durelli, "Recent Advances in the Application of Photoelasticity in the Missile Industry"--October 1962. Two- and three-dimensional photoelastic analysis of grains loaded by pressure and by temperature are presented. Some applications to the optimization of fillet contours and to the redesign of case joints are also included.
4. A. J. Durelli and V. J. Parks, "Experimental Solution of Some Mixed Boundary Value Problems"--April 1964. Means of applying known displacements and known stresses to the boundaries of models used in experimental stress analysis are given. The application of some of these methods to the analysis of stresses in the field of solid propellant grains is illustrated. The presence of the "pinching effect" is discussed.
5. A. J. Durelli, "Brief Review of the State of the Art and Expected Advance in Experimental Stress and Strain Analysis of Solid Propellant Grains"--April 1964. A brief review is made of the state of the experimental stress and strain analysis of solid propellant grains. A discussion of the prospects for the next fifteen years is added.
6. A. J. Durelli, "Experimental Strain and Stress Analysis of Solid Propellant Rocket Motors"--March 1965. A review is made of the experimental methods used to strain-analyze solid propellant rocket motor shells and grains when subjected to different loading conditions. Methods directed at the determination of strains in actual rockets are included.
7. L. Ferrer, V. J. Parks and A. J. Durelli, "An Experimental Method to Analyze Gravitational Stresses in Two-Dimensional Problems"--October 1965. Photoelasticity and moiré methods are used to solve two-dimensional problems in which gravity-stresses are present.

8. A. J. Durelli, V. J. Parks and C. J. del Rio, "Stresses in a Square Slab Bonded on One Face to a Rigid Plate and Shrunk"--November 1965.
A square epoxy slab was bonded to a rigid plate on one of its faces in the process of curing. In the same process the photoelastic effects associated with a state of restrained shrinkage were "frozen-in." Three-dimensional photoelasticity was used in the analysis.
9. A. J. Durelli, V. J. Parks and C. J. del Rio, "Experimental Determination of Stresses and Displacements in Thick-Wall Cylinders of Complicated Shape"--April 1966.
Photoelasticity and moiré are used to analyze a three-dimensional rocket shape with a star shaped core subjected to internal pressure.
10. V. J. Parks, A. J. Durelli and L. Ferrer, "Gravitational Stresses Determined Using Immersion Techniques"--July 1966.
The methods presented in Technical Report No. 7 above are extended to three-dimensions. Immersion is used to increase response.
11. A. J. Durelli and V. J. Parks. "Experimental Stress Analysis of Loaded Boundaries in Two-Dimensional Second Boundary Value Problems"--February 1967.
The pinching effect that occurs in two-dimensional bonding problems, noted in Reports 2 and 4 above, is analyzed in some detail.
12. A. J. Durelli, V. J. Parks, H. C. Feng and F. Chiang, "Strains and Stresses in Matrices with Inserts,"-- May 1967.
Stresses and strains along the interfaces, and near the fiber ends, for different fiber end configurations, are studied in detail.
13. A. J. Durelli, V. J. Parks and S. Uribe, "Optimization of a Slot End Configuration in a Finite Plate Subjected to Uniformly Distributed Load,"--June 1967.
Two-dimensional photoelasticity was used to study various elliptical ends to a slot, and determine which would give the lowest stress concentration for a load normal to the slot length.
14. A. J. Durelli, V. J. Parks and Han-Chow Lee, "Stresses in a Split Cylinder Bonded to a Case and Subjected to Restrained Shrinkage,"--January 1968.
A three-dimensional photoelastic study that describes a method and shows results for the stresses on the free boundaries and at the bonded interface of a solid propellant rocket.
15. A. J. Durelli, "Experimental Stress Analysis Activities in Selected European Laboratories"--August 1968.
This report has been written following a trip conducted by the author through several European countries. A list is given of many of the laboratories doing important experimental stress analysis work and of the people interested in this kind of work. An attempt has been made to abstract the main characteristics of the methods used in some of the countries visited.

16. V. J. Parks, A. J. Durelli and L. Ferrer, "Constant Acceleration Stresses in a Composite Body"--October 1968.
Use of the immersion analogy to determine gravitational stresses in two-dimensional bodies made of materials with different properties.
17. A. J. Durelli, J. A. Clark and A. Kochev, "Experimental Analysis of High Frequency Stress Waves in a Ring"--October 1968.
A method for the complete experimental determination of dynamic stress distributions in a ring is demonstrated. Photoelastic data is supplemented by measurements with a capacitance gage used as a dynamic lateral extensometer.
18. J. A. Clark and A. J. Durelli, "A Modified Method of Holographic Interferometry for Static and Dynamic Photoelasticity"--April 1968.
A simplified absolute retardation approach to photoelastic analysis is described. Dynamic isopachics are presented.
19. J. A. Clark and A. J. Durelli, "Photoelastic Analysis of Flexural Waves in a Bar"--May 1969.
A complete direct, full-field optical determination of dynamic stress distribution is illustrated. The method is applied to the study of flexural waves propagating in a urethane rubber bar. Results are compared with approximate theories of flexural waves.
20. J. A. Clark and A. J. Durelli, "Optical Analysis of Vibrations in Continuous Media"--June 1969.
Optical methods of vibration analysis are described which are independent of assumptions associated with theories of wave propagation. Methods are illustrated with studies of transverse waves in prestressed bars, snap loading of bars and motion of a fluid surrounding a vibrating bar.
21. V. J. Parks, A. J. Durelli, K. Chandrashekhara and T. L. Chen, "Stress Distribution Around a Circular Bar, with Flat and Spherical Ends, Embedded in a Matrix in a Triaxial Stress Field"--July 1969.
A Three-dimensional photoelastic method to determine stresses in composite materials is applied to this basic shape. The analyses of models with different loads are combined to obtain stresses for the triaxial cases.
22. A. J. Durelli, V. J. Parks and L. Ferrer, "Stresses in Solid and Hollow Spheres Subjected to Gravity or to Normal Surface Traction"--October 1969.
The method described in Report No. 10 above is applied to two specific problems. An approach is suggested to extend the solutions to a class of surface traction problems.
23. J. A. Clark and A. J. Durelli, "Separation of Additive and Subtractive Moiré Patterns"--December 1969.
A spatial filtering technique for adding and subtracting images of several gratings is described and employed to determine the whole field of Cartesian shears and rigid rotations.

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24. R. J. Sanford and A. J. Durelli, "Interpretation of Fringes in Stress-Holo-Interferometry"--July 1970.
Errors associated with interpreting stress-holo-interferometry patterns as the superposition of isopachies (with half order fringe shifts) and isochromatics are analyzed theoretically and illustrated with computer generated holographic interference patterns.
25. J. A. Clark, A. J. Durelli and P. A. Laura, "On the Effect of Initial Stress on the Propagation of Flexural Waves in Elastic Rectangular Bars"--December 1970.
Experimental analysis of the propagation of flexural waves in prismatic, elastic bars with and without prestressing. The effects of prestressing by axial tension, axial compression and pure bending are illustrated.
26. A. J. Durelli and J. A. Clark, "Experimental Analysis of Stresses in a Buoy-Cable System Using a Birefringent Fluid"--February 1971.
An extension of the method of photoviscous analysis is presented which permits quantitative studies of strains associated with steady state vibrations of immersed structures. The method is applied in an investigation of one form of behavior of buoy-cable systems loaded by the action of surface waves.
27. A. J. Durelli and T. L. Chen, "Displacements and Finite-Strain Fields in a Sphere Subjected to Large Deformations"--February 1972.
Displacements and strains (ranging from 0.001 to 0.50) are determined in a polyurethane sphere subjected to several levels of diametral compression. A 500 lines-per-inch grating was embedded in a meridian plane of the sphere and moiré effect produced with a non-deformed master. The maximum applied vertical displacement reduced the diameter of the sphere by 27 per cent.
28. A. J. Durelli and S. Machida, "Stresses and Strain in a Disk with Variable Modulus of Elasticity"--March 1972
A transparent material with variable modulus of elasticity has been manufactured that exhibits good photoelastic properties and can also be strain analyzed by moiré. The results obtained suggests that the stress distribution in the disk of variable E is practically the same as the stress distribution in the homogeneous disk. It also indicates that the strain fields in both cases are very different, but that it is possible, approximately, to obtain the stress field from the strain field using the value of E at every point, and Hooke's law.
29. A. J. Durelli and J. Buitrago, "State of Stress and Strain in a Rectangular Belt Pulled Over a Cylindrical Pulley"--June 1972.
Two- and three-dimensional photoelasticity as well as electrical strain gages, dial gages and micrometers are used to determine the stress distribution in a belt-pulley system. Contact and tangential stress for various contact angles and friction coefficients are given.

30. T. L. Chen and A. J. Durelli, "Stress Field in a Sphere Subjected to Large Deformations"--June 1972.
Strain fields obtained in a sphere subjected to large diametral compressions from a previous paper were converted into stress fields using two approaches. First, the concept of strain-energy function for an isotropic elastic body was used. Then the stress field was determined with the Hookean type natural stress-natural strain relation. The results so obtained were also compared.
31. A. J. Durelli, V. J. Parks and H. M. Hasseem, "Helices Under Load"--July 1973.
Previous solutions for the case of close coiled helical springs and for helices made of thin bars are extended. The complete solution is presented in graphs for the use of designers. The theoretical development is correlated with experiments.
32. T. L. Chen and A. J. Durelli, "Displacements and Finite Strain Fields in a Hollow Sphere Subjected to Large Elastic Deformations"--September 1973.
The same methods described in No. 27, were applied to a hollow sphere with an inner diameter one half the outer diameter. The hollow sphere was loaded up to a strain of 30 per cent on the meridian plane and a reduction of the diameter by 20 per cent.
33. A. J. Durelli, H. H. Hasseem and V. J. Parks, "New Experimental Method in Three-Dimensional Elastostatics"--December 1973.
A new material is reported which is unique among three-dimensional stress-freezing materials, in that, in its heated (or rubbery) state it has a Poisson's ratio which is appreciably lower than 0.5. For a loaded model, made of this material, the unique property allows the direct determination of stresses from strain measurements taken at interior points in the model.
34. J. Wolak and V. J. Parks, "Evaluation of Large Strains in Industrial Applications"--April 1974.
It was shown that Mohr's circle permits the transformation of strain from one axis of reference to another, irrespective of the magnitude of the strain, and leads to the evaluation of the principal strain components from the measurement of direct strain in three directions.
35. A. J. Durelli, "Experimental Stress Analysis Activities in Selected European Laboratories"--April 1975.
Continuation of Report No. 15 after a visit to Belgium, Holland, Germany, France, Turkey, England and Scotland.
36. A. J. Durelli, V. J. Parks and J. O. Bühler-Vidal, "Linear and Non-linear Elastic and Plastic Strains in a Plate with a Big Hole Loaded Axially in its Plane"--July 1975.
Strain analysis of the ligament of a plate with a big hole indicates that both geometric and material non-linearity may take place. The strain concentration factor was found to vary from 1 to 2 depending on the level of deformation.

37. A. J. Durelli, V. Pavlin, J. O. Bühler-Vidal and G. Ome, "Elastostatics of a Cubic Box Subjected to Concentrated Loads"--August 1975.
Analysis of experimental strain, stress and deflection of a cubic box subjected to concentrated loads applied at the center of two opposite faces. The ratio between the inside span and the wall thickness was varied between approximately 5 and 121.
38. A. J. Durelli, V. J. Parks and J. O. Bühler-Vidal, "Elastostatics of Cubic Boxes Subjected to Pressure"--March 1976.
Experimental analysis of strain, stress and deflections in a cubic box subjected to either internal or external pressure. Inside span-to-wall thickness ratio varied from 5 to 14.
39. Y. Y. Hung, J. D. Hovanesian and A. J. Durelli, "New Optical Method to Determine Vibration-Induced Strains with Variable Sensitivity After Recording"--November 1976.
A steady state vibrating object is illuminated with coherent light and its image slightly misfocused. The resulting specklegram is "time-integrated" as when Fourier filtered gives derivatives of the vibrational amplitude.
40. Y. Y. Hung, C. Y. Liang, J. D. Hovanesian and A. J. Durelli, "Cyclic Stress Studies by Time-Averaged Photoelasticity"--November 1976.
"Time-averaged isochromatics" are formed when the photographic film is exposed for more than one period. Fringes represent amplitudes of the oscillating stress according to the zeroth order Bessel function.
41. Y. Y. Hung, C. Y. Liang, J. D. Hovanesian and A. J. Durelli, "Time-Averaged Shadow Moiré Method for Studying Vibrations"--November 1976.
Time-averaged shadow moiré permits the determination of the amplitude distribution of the deflection of a steady vibrating plate.
42. J. Buitrago and A. J. Durelli, "On the Interpretation of Shadow-Moiré Fringes"--April 1977.
Possible rotations and translations of the grating are considered in a general expression to interpret shadow-moiré fringes and on the sensitivity of the method. Application to an inverted perforated tube.
43. J. der Hovanesian, "18th Polish Solid Mechanics Conference." Published in European Scientific Notes of the Office of Naval Research, in London, England, Dec. 31, 1976.
Comments on the planning and organization of, and scientific content of paper presented at the 18th Polish Solid Mechanics Conference held in Wisla-Jawornik from September 7-14, 1976.
44. A. J. Durelli, "The Difficult Choice,"--May 1977.
The advantages and limitations of methods available for the analyses of displacements, strain, and stresses are considered. Comments are made on several theoretical approaches, in particular approximate methods, and attention is concentrated on experimental methods: photoelasticity, moiré, brittle and photoelastic coatings, gages, grids, holography and speckle to solve two- and three-dimensional problems in elasticity, plasticity, dynamics and anisotropy.

45. C. Y. Liang, Y. Y. Hung, A. J. Durelli and J. D. Hovanesian, "Direct Determination of Flexural Strains in Plates Using Projected Gratings,"--June 1977.
The method requires the rotation of one photograph of the deformed grating over a copy of itself. The moiré produced yields strains by optical double differentiation of deflections. Applied to projected gratings the idea permits the study of plates subjected to much larger deflections than the ones that can be studied with holograms.
46. A. J. Durelli, K. Brown and P. Yee, "Optimization of Geometric Discontinuities in Stress Fields"--March 1978.
The concept of "coefficient of efficiency" is introduced to evaluate the degree of optimization. An ideal design of the inside boundary of a tube subjected to diametral compression is developed which decreases its maximum stress by 25%, at the time it also decreases its weight by 10%. The efficiency coefficient is increased from 0.59 to 0.95. Tests with a brittle material show an increase in strength of 20%. An ideal design of the boundary of the hole in a plate subjected to axial load reduces the maximum stresses by 26% and increases the coefficient of efficiency from 0.54 to 0.90.
47. J. D. Hovanesian, Y. Y. Hung and A. J. Durelli, "New Optical Method to Determine Vibration-Induced Strains With Variable Sensitivity After Recording"--May 1978.
A steady-state vibrating object is illuminated with coherent light and its image is slightly misfocused in the film plane of a camera. The resulting processed film is called a "time-integrated specklegram." When the specklegram is Fourier filtered, it exhibits fringes depicting derivatives of the vibrational amplitude. The direction of the spatial derivative, as well as the fringe sensitivity may be easily and continuously varied during the Fourier filtering process. This new method is also much less demanding than holographic interferometry with respect to vibration isolation, optical set-up time, illuminating source coherence, required film resolution. etc.
48. Y. Y. Hung and A. J. Durelli, "Simultaneous Determination of Three Strain Components in Speckle Interferometry Using a Multiple Image Shearing Camera,"--September 1978
This paper describes a multiple image-shearing camera. Incorporating coherent light illumination, the camera serves as a multiple shearing speckle interferometer which measures the derivatives of surface displacements with respect to three directions simultaneously. The application of the camera to the study of flexural strains in bent plates is shown, and the determination of the complete state of two-dimensional strains is also considered. The multiple image-shearing camera uses an interference phenomena, but is less demanding than holographic interferometry with respect to vibration isolation and the coherence of the light source. It is superior to other speckle techniques in that the obtained fringes are of much better quality.

49. A. J. Durelli and K. Rajaiah, "Quasi-square Hole With Optimum Shape in an Infinite Plate Subjected to In-plane Loading"--January 1979. This paper deals with the optimization of the shape of the corners and sides of a square hole, located in a large plate and subjected to in-plane loads. Appreciable disagreement has been found between the results obtained previously by other investigators. Using an optimization technique, the authors have developed a quasi-square shape which introduces a stress concentration of only 2.54 in a uniaxial field, the comparable value for the circular hole being 3. The efficiency factor of the proposed optimum shape is 0.90, whereas the one of the best shape developed previously was 0.71. The shape also is developed that minimizes the stress concentration in the case of biaxial loading when the ratio of biaxiality is 1:-1.
50. A. J. Durelli and K. Rajaiah, "Optimum Hole Shapes in Finite Plates Under Uniaxial Load,"--February 1979. This paper presents optimized hole shapes in plates of finite width subjected to uniaxial load for a large range of hole to plate widths (D/W) ratios. The stress concentration factor for the optimized holes decreased by as much as 44% when compared to circular holes. Simultaneously, the area covered by the optimized hole increased by as much as 26% compared to the circular hole. Coefficients of efficiency between 0.91 and 0.96 are achieved. The geometries of the optimized holes for the D/W ratios considered are presented in a form suitable for use by designers. It is also suggested that the developed geometries may be applicable to cases of rectangular holes and to the tip of a crack. This information may be of interest in fracture mechanics.
51. A. J. Durelli and K. Rajaiah, "Determination of Strains in Photoelastic Coatings,"--May 1979. Photoelastic coatings can be cemented directly to actual structural components and tested under field conditions. This important advantage has made them relatively popular in industry. The information obtained, however, may be misinterpreted and lead to serious errors. A correct interpretation requires the separation of the principal strains and so far, this operation has been found very difficult. Following a previous paper by one of the authors, it is proposed to drill small holes in the coating and record the birefringence at points removed from the edge of the holes. The theoretical background of the method is reviewed; the technique necessary to use it is explained and two applications are described. The precision of the method is evaluated and found satisfactory in contradiction to information previously published in the literature.

OPTIMIZED INNER BOUNDARY SHAPES IN CIRCULAR RINGS

UNDER DIAMETRAL COMPRESSION

by

A. J. Durelli and K. Rajaiah

ABSTRACT

Using a method developed by the authors, the configuration of the inside boundary of circular rings, subjected to diametral compression, has been optimized, keeping cleared the space enclosed by the original circular inside boundary. The range of diameters studied was $0.33 \leq ID/OD \leq 0.7$. In comparison with circular rings of the same ID/OD, the stress concentrations have been reduced by about 30%, the weight has been reduced by about 10% and coefficients of efficiency of about 0.96 have been attained. The maximum values of compressive and tensile stresses on the edge of the hole, are approximately equal, there are practically no gradients of stress along the edge of the hole, and sharp corners exhibit zero stress. The geometries for each ID/OD design are given in detail.

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INTRODUCTION

Consider a circular ring with outer diameter OD and inner diameter ID subjected to a diametral compressive load P . As a classical problem in elasticity, theoretical solutions have been found for this problem by Timoshenko and others⁽¹⁾. The problem has several practical applications in tunnel, roller and pipe designs and many experimental investigations have also been reported in literature^{(2),(3)}. Stresses in a thick cylinder having a non-circular hole, when subjected to diametral compression, were perhaps first analyzed by Seika⁽⁴⁾. Using the complex-variable approach, he presented numerical results for some typical quasi-square holes in a cylinder with a near circular outer boundary. However, it does not appear to have been realized till recently⁽⁵⁾ that the stress concentration factor (s.c.f.) on the inner boundary can be very effectively brought down and the hole shape optimized by suitably changing the inner boundary of the ring. Durelli, Brown and Yee⁽⁵⁾ showed for the first time that simple photoelastic experiments can be utilized for this purpose and the removal of material from low stress regions around holes in any stress field leads to optimum shapes for the hole boundaries. In the present paper, the same approach is followed and optimized inner boundary shapes for circular rings with different ID/OD ratios are presented.

CONSTRAINS OF THE PROBLEM

For the optimization process, the following constrains were stipulated:

- a) the outside boundary has to be kept circular with diameter OD, b) the inside boundary has to clear a circle of given diameter ID, and c) the allowable maximum stress for tension is about the same as for compression.

METHOD

The optimization process requires the use of two dimensional photo-elastic models, loaded in a large field polariscope. The removal of material from the low stress regions around the hole is obtained by careful filing till an isochromatic fringe coincides with the boundary in the tensile and compressive regions respectively. The constrains of the problem dictate the amount of material that may be removed.

It was proposed in an earlier paper⁽⁵⁾ and used subsequently in other optimization studies^{(6),(7)} that the degree of optimization be evaluated quantitatively by a coefficient of efficiency k_{eff} defined as

$$k_{eff} = \frac{1}{S_2 - S_0} \left\{ \frac{\int_{S_0}^{S_1} \sigma_t^+ ds}{\sigma_{all}^+} + \frac{\int_{S_1}^{S_2} \sigma_t^- ds}{\sigma_{all}^-} \right\}$$

where σ_{all} represents the maximum allowable stress (the positive and negative superscripts referring to tensile and compressive stresses, respectively), S_0 and S_1 are the limiting points of the segment of boundary subjected to tensile stresses and S_1 and S_2 are the limiting points of the segment of boundary with compressive stresses.

The significance of the coefficient of efficiency was discussed in Refs. (5) and (6). The same criterion has been used in the present work to evaluate the optimized hole shapes.

EXPERIMENTAL PROCEDURE

Two dimensional photoelastic experiments were conducted with 0.23 in. (5.8 mm) thick Homalite-100 plates (fringe constant of 133.2 lb/in-fr (23.3 kn/m-fr)). The outer diameter of the ring was maintained at 5 in. (127 mm) while the inner diameter was progressively increased. Optimization of the inner boundary was carried out for ID/OD = 0.33, 0.43, 0.53, 0.63 and 0.70 with the models subjected to a diametral compressive load P . Material was removed from low stress regions as explained above. To improve the control of the filing process, a binocular magnifier with a set of polarizer and quarter wave plates attached to each of its lenses was used.

RESULTS

The isochromatic patterns for two typical hole shapes are shown in Figs. 1 and 2. The stress distributions around the optimized boundaries for the ID/OD ratios considered are presented in Fig. 3. Information regarding the stress distribution around circular holes in rings with the same ID/OD ratios is also included in the same figure for comparison purposes. Seika's theoretical results⁽⁴⁾ for two ID/OD ratios which are relevant to the stress distribution considered are also included. The stress concentration factor (s.c.f.) for the tensile and compressive regions of the optimized holes for different ID/OD ratios are plotted in Fig. 4 and are compared with those for the circular holes.

The empirically developed optimum geometries of the inner boundary have been fitted with a combination of circles of different diameters and common tangents at the points of intersection. The inner hole geometries for the different ID/OD ratios are shown in Fig. 5.

By consolidating the information regarding the geometries, the different radii of curvature for the optimized hole edges are presented in graphical form in Fig. 6 in a way which can be directly used by designers.

DISCUSSION

The isochromatic patterns in Figs. 1 and 2 show that the inner boundaries are well optimized with an isochromatic fringe following the boundary very closely. Fig. 1 for ID/OD = 0.43 shows that optimization of the inner boundary has not affected the stress distribution on the outer circular boundary. Fig. 2 for ID/OD = 0.7 shows that optimization of the inner boundary has lead to the optimization of the outer boundary as well with a small increase in the stress on the outer boundary. The results indicate that such favorable situations occur for ID/OD > 0.53.

The stress distributions around the optimized boundaries given in Fig. 3 show that as the ID/OD ratio increases, the s.c.f. also increases. The increase is, however, much smaller for the optimized shapes than for the circular rings. Comparison of Seika's results for the two ID/OD ratios brings out the fact that he could get more favorable stress distributions than those for a circular ring at least on the tensile segment of the boundary for ID/OD = 0.32 and on the compressive segment for ID/OD = 0.344. Due to the limitations of the mapping function used by him, his method cannot yield hole shapes which are favorable from the stress point of view both on the tensile and compressive portions of the boundary.

The s.c.f. for the tensile and compressive portions of the boundary for the circular and optimized rings in Fig. 4 show that the optimization has lead to a significant reduction in stresses. The reduction in the tensile concentration factor is much larger than the reduction in the compressive concentration factor. The reduction in s.c.f. is highest for $ID/OD = 0.33$, decreases slightly and then increases again beyond $ID/OD = 0.53$. The reduction in weight of the ring increases with increase in ID/OD approximately in a linear way. The coefficient of efficiency k_{eff} for all the optimized holes is about 0.95, the corresponding k_{eff} for circular rings being about 0.61.

The optimized hole geometries in Fig. 5 show that the hole shapes have similar appearance with changes only in the radii of curvature. For $ID/OD = 0.43$, a shape very close to the optimum can be obtained using a radius equal to $(R_o + R_i)$ for both the tensile and compressive boundaries. The information on the radii of the elements of the holes given in Figs. 5 and 6 indicates that for small hole sizes, the optimized hole shape would be the same as the one in an infinite plate under uniaxial loading⁽⁶⁾ with the tensile edge becoming a straight line.

ACKNOWLEDGMENTS

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References

1. Timoshenko S., "On the Distribution of Stresses in a Circular Ring Compressed by Two Forces Along a Diameter", Phil. Mag., Vol. 14, pp. 1014-1019, 1922. (Peterson in "Stress Concentration Factors", Wiley 1974, considers other theoretical and experimental contributions).
2. Heywood R. N., "Photoelasticity for Designers", Pergamon Press 1969.
3. Durelli A. J. and Parks V. J., "Moire Analysis of Strain", Prentice-Hall, 1970.
4. Seika M., "The Stresses in a Thick Cylinder Having a Square Hole Under Concentrated Loading", J. Appl. Mech., Vol. 25, pp. 571-574, 1958.
5. Durelli A. J., Brown K. and Yee P., "Optimization of Geometric Discontinuities in Stress Fields", Exp. Mech., Vol. 18, 1978, pp. 303-308.
6. Durelli A. J. and Rajaiah K., "Quasi-square Hole with Optimum Shape in an Infinite Plate Subjected to In-plane Loading", ONR Report No. 49, Oakland University, Jan. 1979.
7. Durelli A. J. and Rajaiah K., "Optimum Hole Shapes in Finite Plates under Uniaxial Load", ONR Report No. 50, Oakland University, Feb. 1979.

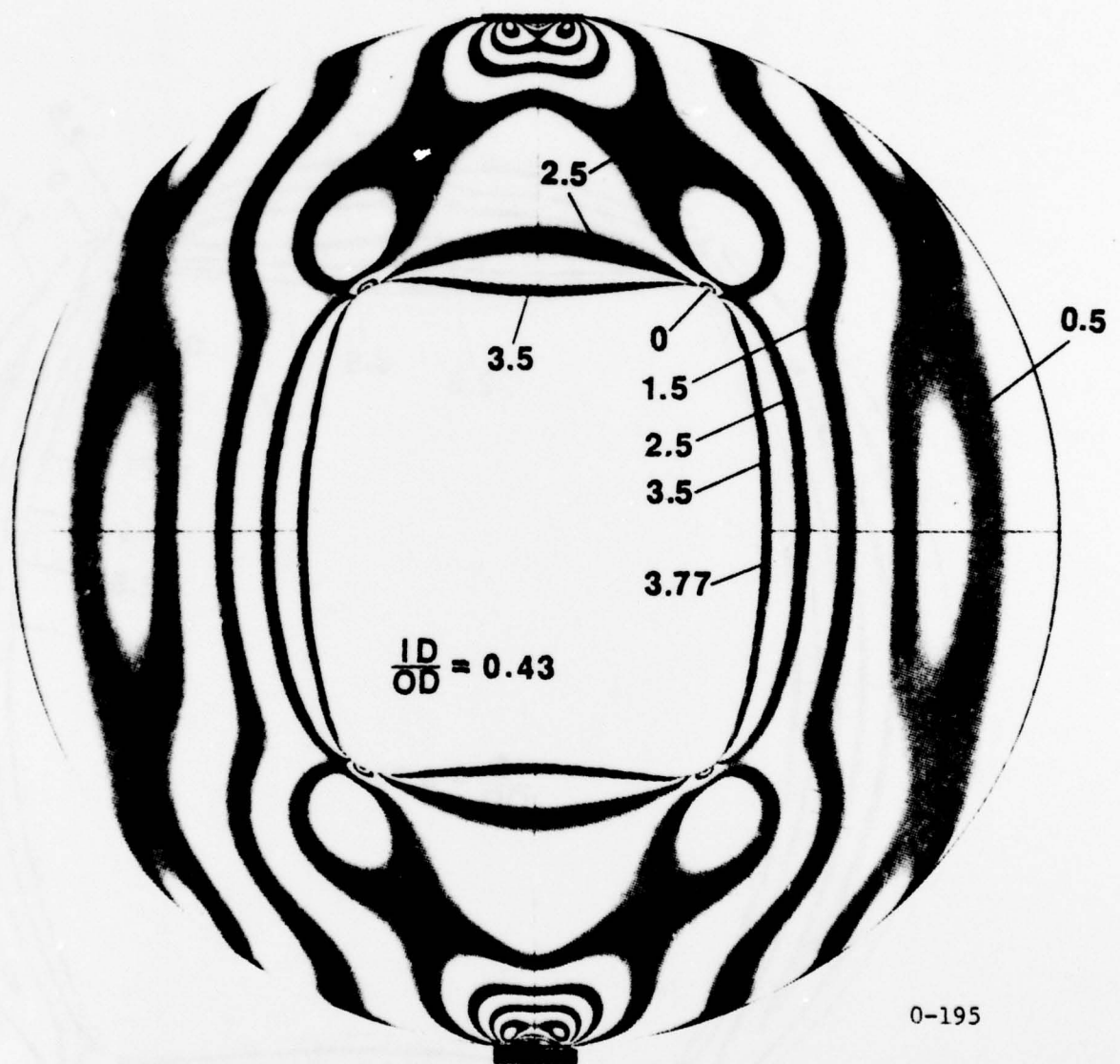


FIG. 1 ISOCHROMATICS IN A CIRCULAR RING WITH OPTIMIZED INNER BOUNDARY, WHEN SUBJECTED TO DIAMETRAL COMPRESSION
 $ID/OD = 0.43$

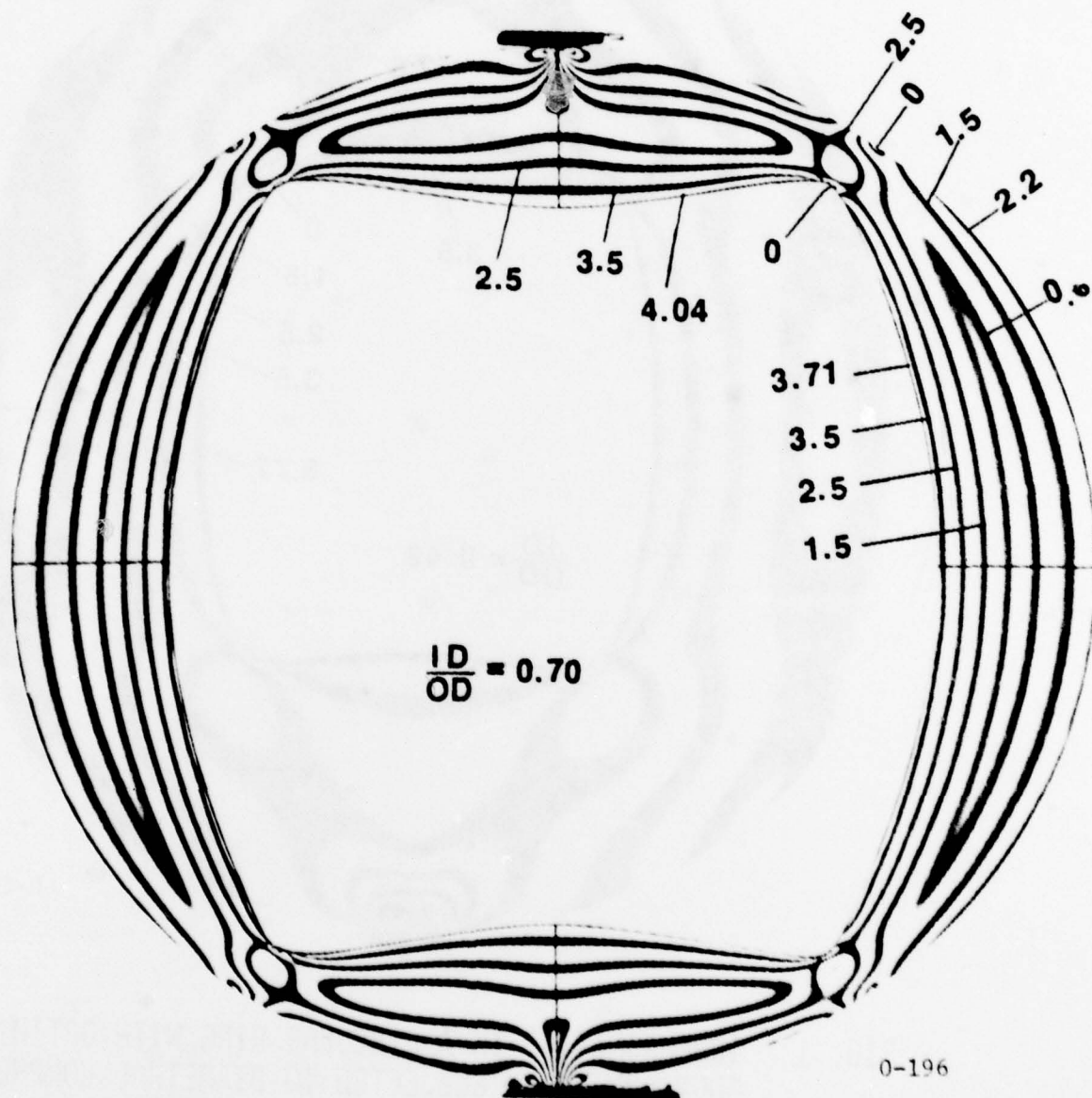


FIG. 2 ISOCHROMATICS IN A CIRCULAR RING WITH OPTIMIZED INNER BOUNDARY, WHEN SUBJECTED TO DIAMETRAL COMPRESSION
 $ID/OD = 0.70$

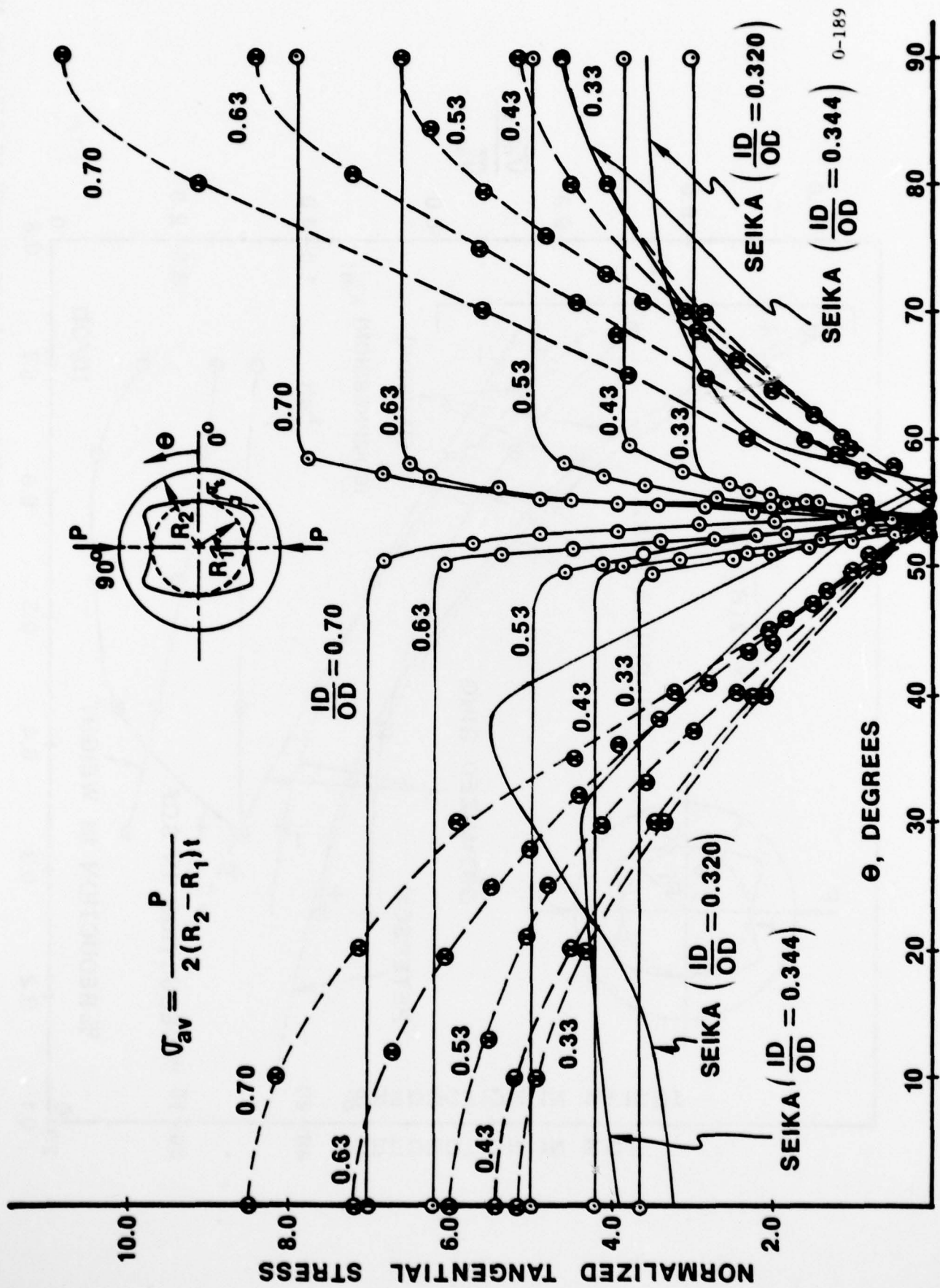


FIG. 3 STRESS DISTRIBUTION ALONG THE EDGE OF THE OPTIMIZED INNER BOUNDARY OF CIRCULAR RINGS SUBJECTED TO DIAMETRAL COMPRESSION

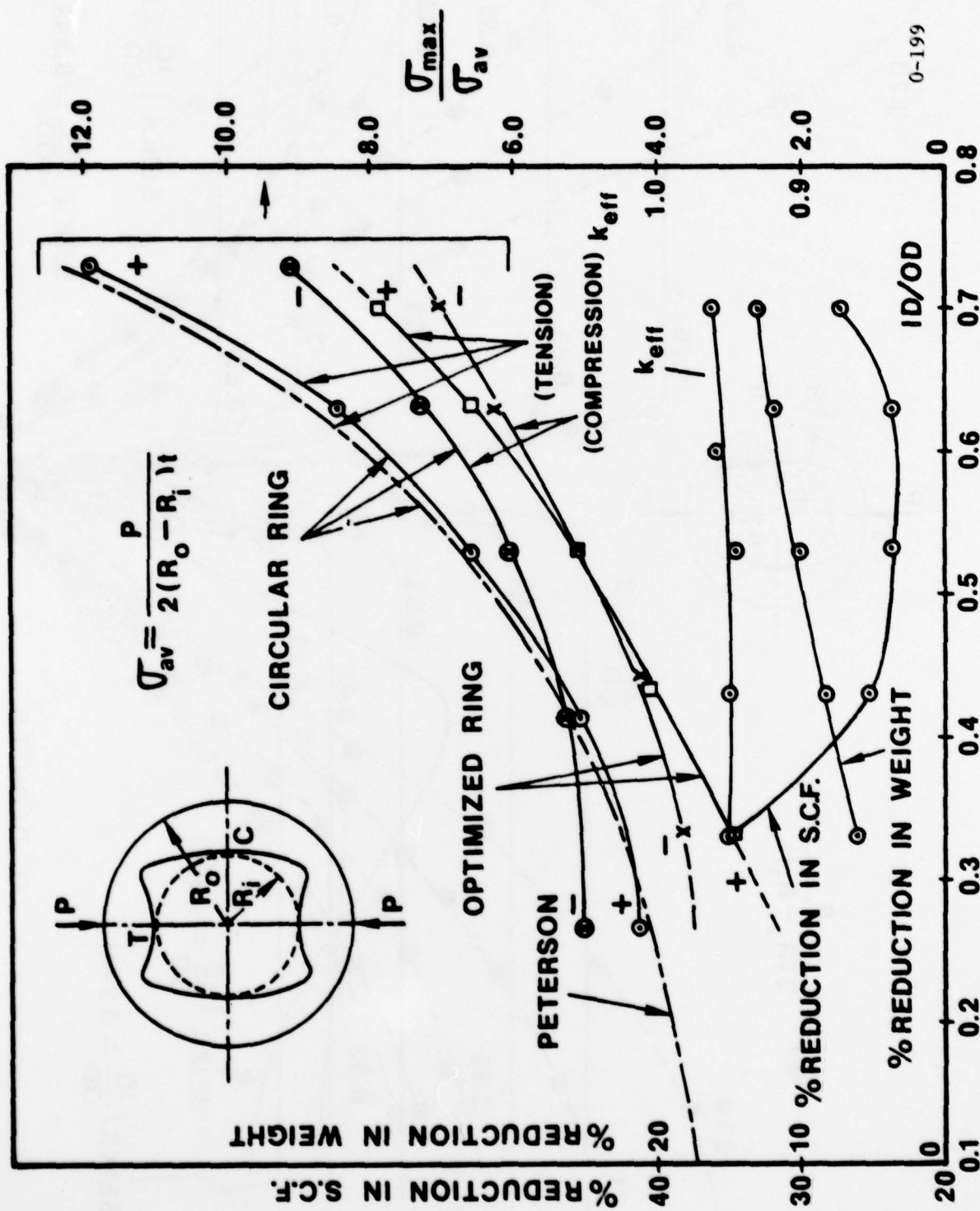
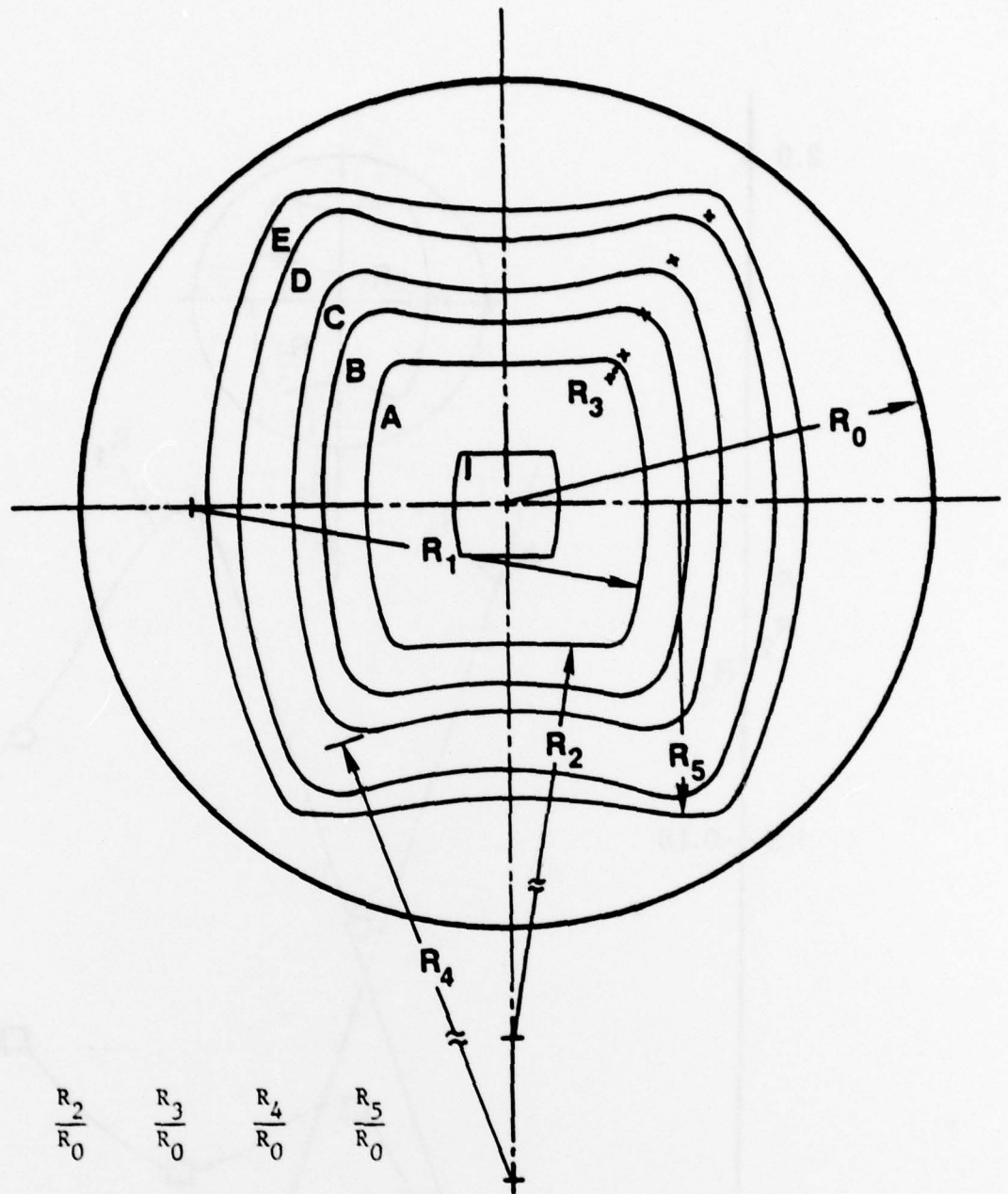


FIG. 4 STRESS CONCENTRATION FACTORS, WEIGHT AND COEFFICIENT OF EFFICIENCY IN CIRCULAR AND OPTIMIZED RINGS, SUBJECTED TO DIAMETRAL COMPRESSION.



	$\frac{ID}{OD}$	$\frac{R_1}{R_0}$	$\frac{R_2}{R_0}$	$\frac{R_3}{R_0}$	$\frac{R_4}{R_0}$	$\frac{R_5}{R_0}$
A	0.33	1.06	1.71	0.05	-	-
B	0.43	1.43	1.44	0.11	-	-
C	0.53	1.73	1.01	0.10	1.25	-
D	0.63	1.51	1.13	0.12	-	-
E	0.70	1.49	1.49	0.06	-	0.73
I	0.14	0.35	∞	0.05	-	- (Estimated)

0-200

FIG. 5 OPTIMUM SHAPE OF THE INNER BOUNDARY OF CIRCULAR RINGS
SUBJECTED TO DIAMETRAL COMPRESSION.

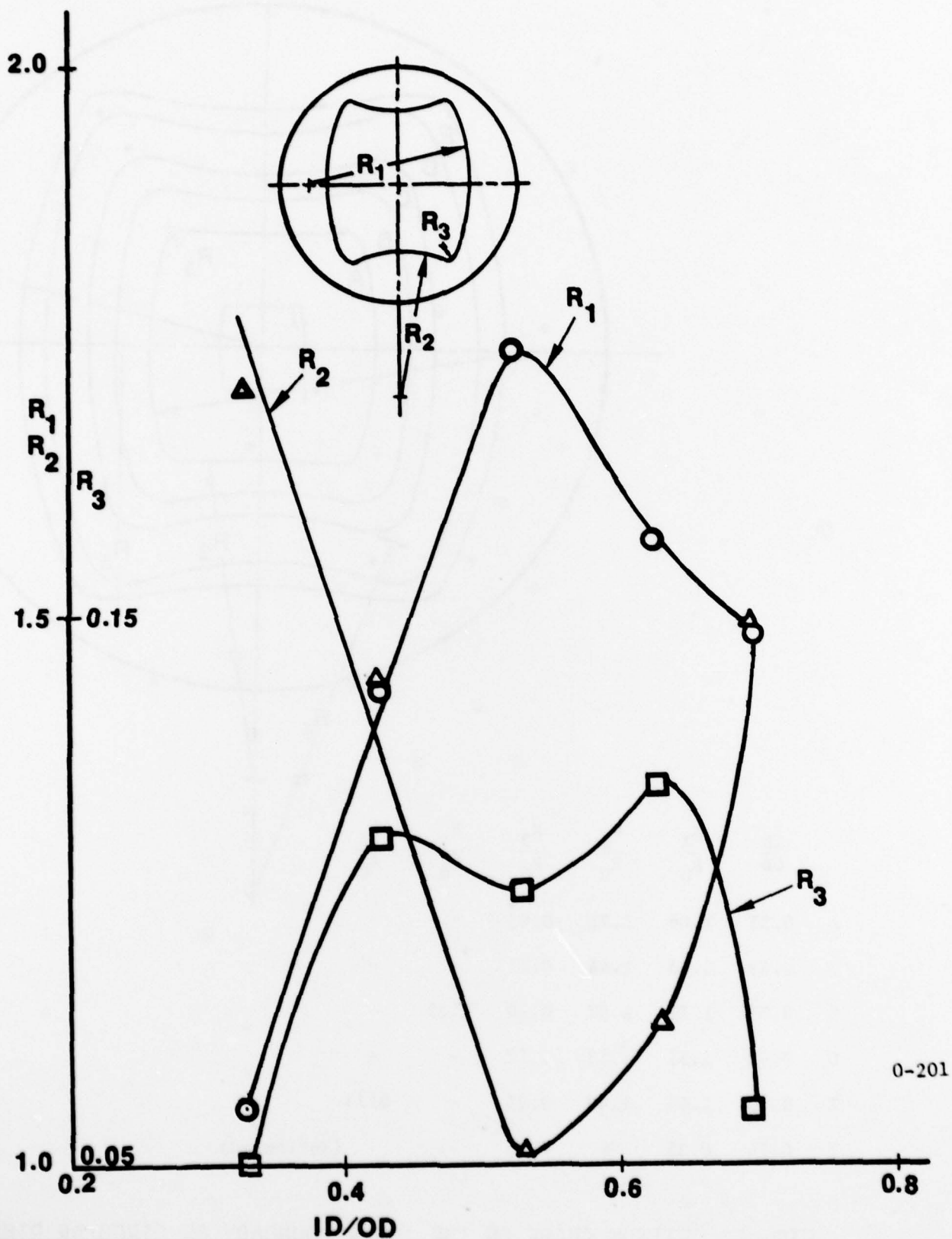


FIG. 6 RADII OF THE ELEMENTS OF HOLES PRODUCING OPTIMUM DISTRIBUTION OF STRESS IN CIRCULAR RINGS UNDER DIAMETRAL COMPRESSION.

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20. ABSTRACT (Continue on reverse side if necessary and identify by block number) Using a method developed by the authors, the configuration of the inside boundary of circular rings, subjected to diametral compression, has been optimized, keeping cleared the space enclosed by the original circular inside boundary. The range of diameters studied was $0.33 < ID/OD < 0.7$. In comparison with circular rings of the same ID/OD, the stress concentrations have been reduced by about 30%, the weight has been reduced by about 10% and coefficients of efficiency of about 0.96 have been attained. The		

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maximum values of compressive and tensile stresses on the edge of the hole, and sharp corners exhibit zero stress. The geometries for each ID/OD design are given in detail.

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